

(12) UK Patent Application (19) GB (11) 2 086 983 A

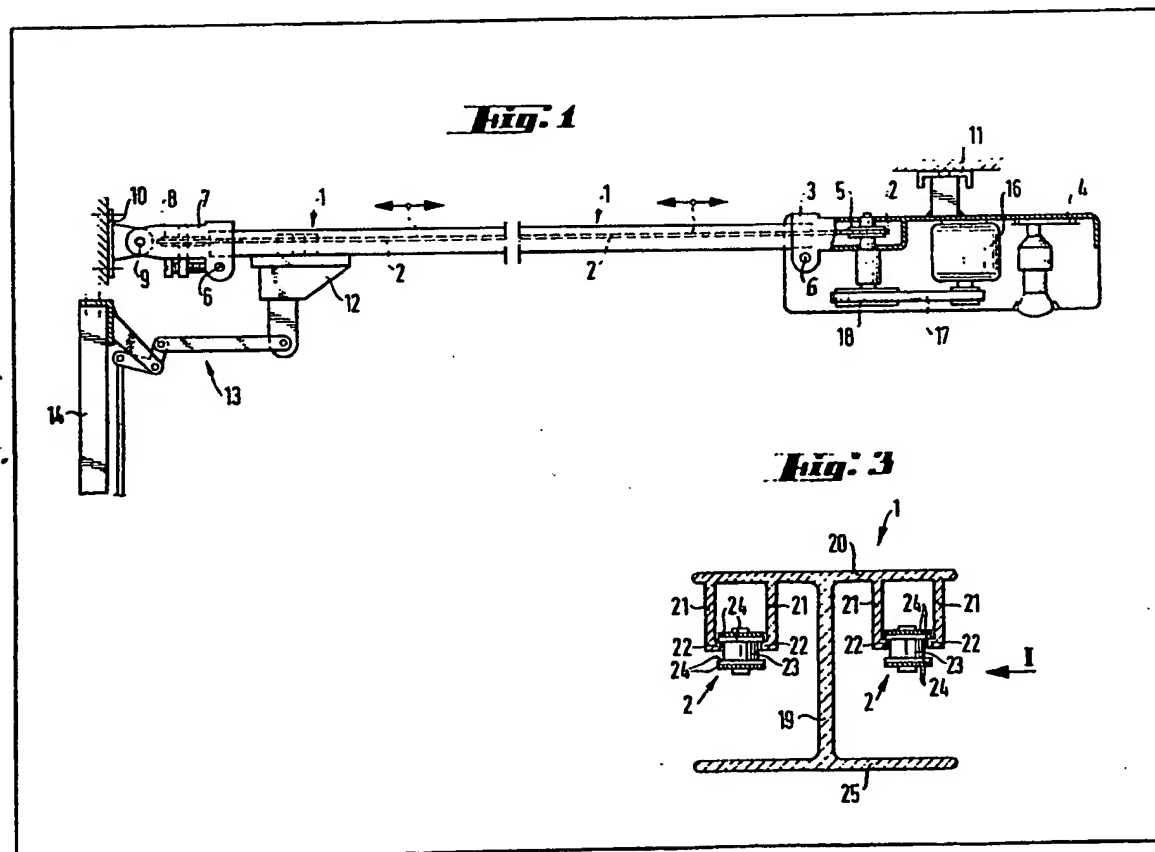
(21) Application No 8133035
 (22) Date of filing 3 Nov 1981
 (30) Priority data
 (31) 8029666U
 (32) 6 Nov 1980
 (33) Fed. Rep. of Germany (DE)
 (43) Application published
 19 May 1982
 (51) INT CL³
 E05F 15/10
 (52) Domestic classification
 E2M 11D1 12A 13 16
 (56) Documents cited
 GB 2048368A
 GB 2043958A
 GB 710520
 (58) Field of search
 E2M
 (71) Applicants
 Michael Hormann trading as
 Hormann KG,
 4834 Harsewinkel, West
 Germany, Federal
 Republic of Germany
 (72) Inventor
 Michael Hormann

(74) Agent
 J. A. Boutland,
 8 Heatherstone Avenue,
 Hythe, Southampton
 SO4 5LQ

(54) Supporting lengths of chain in
 draw chain drives

(57) A motor drive arrangement for a
 door 14 has a draw chain 2 which is
 guided between a drive chain wheel 5

and a guide chain wheel 8, the chain 2
 being connected to the door 14 by
 way of a draw member 12 movable by
 the chain along a mounting rail 1 which
 is provided with vertical webs 21
 which extend longitudinally of the rail
 and have intumed flanges 22 which
 form between them gaps for receiving
 the pins or rollers 23 of the chain 2.
 Above and below the flanges 22,
 spaces which are wider than the gaps
 receive the side plates 24 of the chain.



GB 2 086 983 A

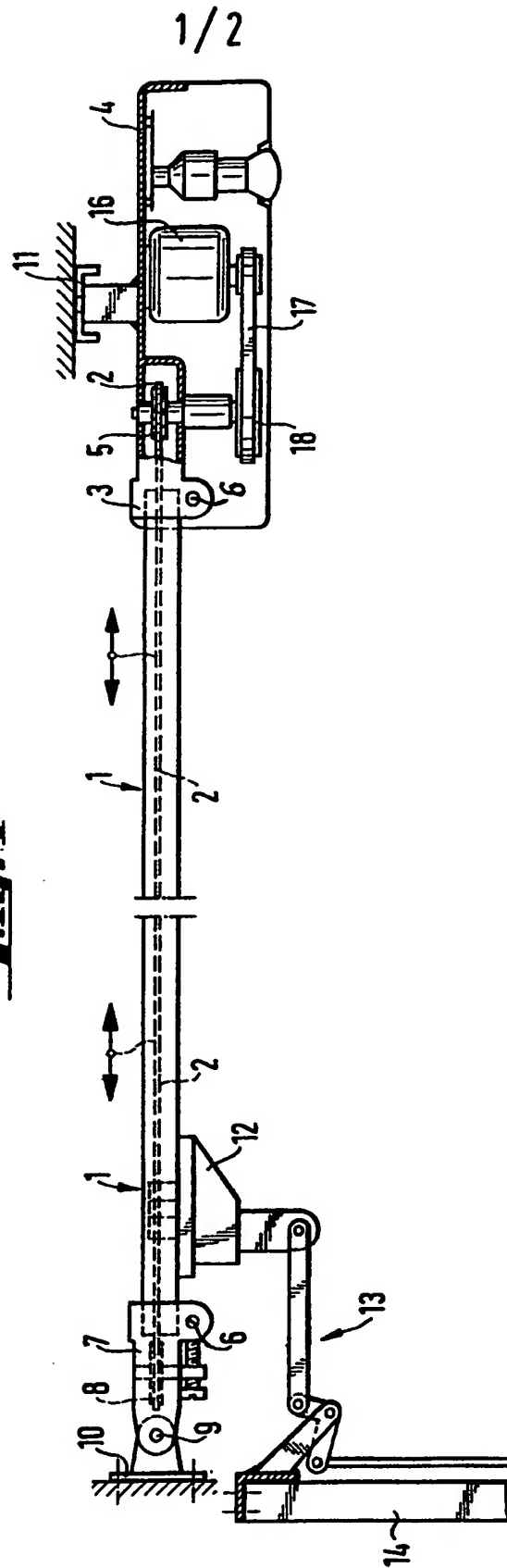
Fig. 1

Fig. 2

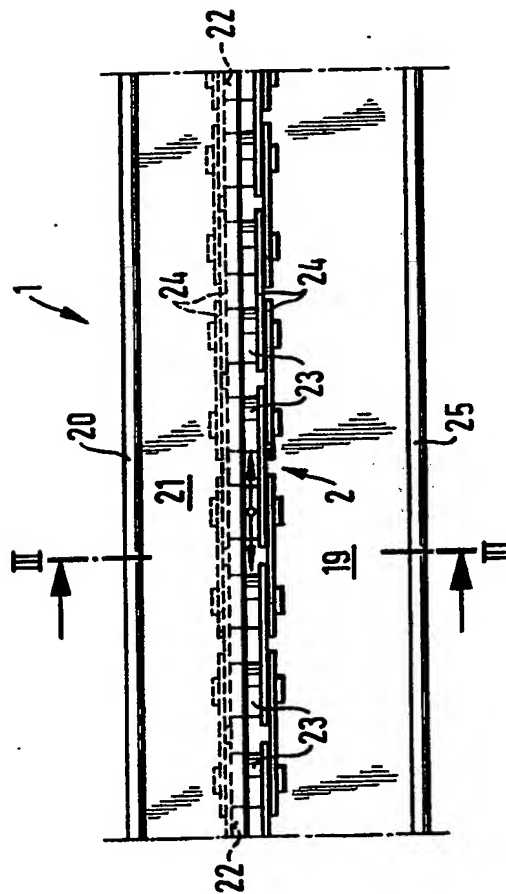
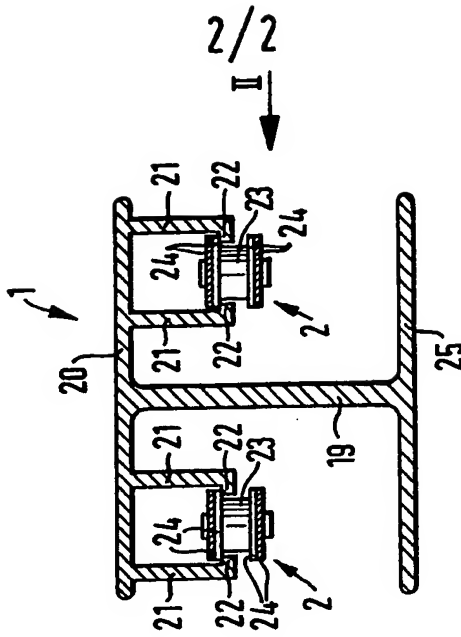


Fig. 3



SPECIFICATION

Improvements in or relating to draw chain drive arrangements

This invention relates to draw chain drive
5 arrangements.

Draw chain drive arrangements of the kind with
which the present invention is concerned are
those wherein a draw member which is secured to
the chain is guided by means of the chain along a
10 straight path over which the chain has hitherto in
principle been guided in a self-supporting manner.
The endless chain which, from an ideal point of
view, circulates in a horizontal plane, accordingly
sags in the axial direction of its links over the quite
15 long conveyor sections outside of the positions at
which it is secured to the draw member. In
addition, it moves with corresponding ease about
its link pivots, transversely with respect to the
longitudinal direction of the runs of the chain. In
20 operation, such transverse movements of the
chain, which may also be in the form of a swinging
motion, may be comparatively severe.

The movements of the chain are not harmful as
long as the chain does not strike against another
25 object anywhere and in particular as long as the
motion of the chain onto the chain wheels is not
detrimentally affected by the movements of the
chain. It will be appreciated that it is necessary to
reckon on the provision of a relatively large
30 amount of space for the chain to move, in order to
ensure that the chain does not strike against
another object, with troublesome consequences.
In many cases, it is found to be necessary for a
switch, for example, a limit switch, to be actuated
35 by means of a switch actuating member which is
secured to the chain, in dependence on the
conveying position of the chain. If such switch
actuation occurs outside the region of the chain
wheels, the chain is guided separately in order to
40 be able correctly to carry out the switch actuating
operation, over a short distance. Chain guide
means of this kind, outside the region of the chain
wheels, in the form of slide guide means, are
known for example from German utility model No.
45 78 17 731. The slide guide means used therein
are guide members of plastics material, which are
restricted to a very short longitudinal region of the
chain. Such guide means are installed not only in
front of contacts, but also in the run-in region in
50 front of the chain wheels.

The short length of the plastics guide members
which engage around the upper side plates of the
chain in the manner of clip means are intended to
produce the minimum possible amount of friction
55 between the chain and the guide member. More
particularly, the guide members must "lift up" the
chain which sags down in the rest of the run of the
chain, and in particular must carry the forces
occurring due to the transverse movements. The
60 normal forces which are thus applied to the guide
members are relatively high so that, even with a
low coefficient of friction, considerable friction
losses still occur.

Mounting the guide members on the mounting

65 rail involves corresponding assembly expenditure,
and this applies even when the mounting rail of
the known arrangement already has a continuous
T-shaped structural portion for fixing the guide
members, in order more easily to fit the guide
70 members or to adjust them to the proper position.

The present invention seeks to provide a chain
guide means in the region between the chain
wheels, which is precise and in particular simple,
in regard to the expenditure to be involved in that
75 respect, more particularly in connection with
assembly thereof.

According to the invention, this is achieved by
the subject-matter recited in Claim 1.

The construction of the mounting rail according
80 to the invention provides that the chain is guided
virtually along its entire path between the chain
wheels around which the chain passes, and the
guide means prevents the chain both from sagging
under its own weight and also moving
85 transversely in a horizontal plane, and in particular
does not permit such movements to escalate into
an oscillating motion. This therefore prevents the
dynamic forces which are produced by such
movements, and thus eliminates frictional forces
90 along the chain. By virtue of the chain being
supported at a plurality of positions in the region
of each side plate on the guide means, the
pressure loadings are low. Even when the guide
means is made from the same material as the
95 mounting rail, for example in the form of a one-
part hollow shaped member in the form of a
preferred embodiment of the mounting rail
together with the guide means, the frictional
resistance forces which occur are generally low.
100 The guide action is very precise and particularly
exact at each point on the run of the chain,
whereby in particular there is no need for
mounting the guide means at the right position or
adjusting the position of the guide means, as was
105 required in the state of the art. The production
cost of the guide means is at a particularly low
level due to the guide means being formed in one
piece with the mounting rail, and, in a preferred
embodiment in the form of an extruded member,
110 is even negligibly low.

The bar portions which engage between the
side plates of the chain and which engage against
the rollers thereof may be connected to the rest of
the configuration of the mounting rail in many
115 ways. Preferably, the arrangement is such as to
ensure that a hollow shaped member is formed
over the upper side plates of the chain so that the
chain is protected as viewed from above.

Further features of the invention are set out in
120 the subsidiary claims, in conjunction with the
embodiment about to be described by way of
example only, with reference to the accompanying
drawings, wherein:—

Figure 1 is a side view of a draw chain drive
125 arrangement, including the mounting rail,

Figure 2 is a side view of a longitudinal portion
of the mounting rail with a portion of
corresponding length of draw chain, and

Figure 3 is a view in cross-section, taken along

lines III—III of Figure 1.

The overall view of the draw chain arrangement shown in Figure 1 illustrates, in a broad breakdown, the mounting rail 1 along which the chain 2 is guided. The mounting rail 1, which comprises an aluminium alloy, is fixed at both ends, at one end by a connecting member 3 forming part of a housing member 4. The housing member 4 rotatably supports a drive pinion 5 for the chain 2. The connecting member 3 is fixed to the housing member 4 by means of clamp screw 6. The other end of the rail 1 is fixed to a connecting member 7 which rotatably supports a direction-changing pinion 8 for the chain 2. The connecting member 7 is connected by way of a pivot arrangement 9 to a wall fixing plate 10 so that there is no need, on installing the arrangement, to achieve precise alignment with respect to the wall fixing means 11 of the housing member 4. Fixed to the chain 2 is a draw member 12 which, by way of a draw assembly generally indicated by reference numeral 13, forms a connection to a movable door member 14. The door member 14 is part of a garage door assembly.

Secured to the housing member 4 is an electric motor 16 which acts on the drive chain wheel 5 by way of a drive belt 17 and a belt pulley 18. Depending on the direction of rotation of the motor 16, the chain 2 is moved by way of the drive chain wheel 5 in one direction or the other. Thus the draw member 12 is moved along the mounting rail 1 in the corresponding direction, whereby the movable door member 14 (of which only part is shown) is lifted out of or lowered into the region of the garage door opening, by way of the draw assembly 13, with actuation of a locking catch linkage which is included in Figure 1.

The side view of the longitudinal portion of the mounting rail 1 shown by Figure 2 and the cross-section through the rail 1 shown by Figure 3 illustrate that an extruded shaped member having the basic configuration of an "I" is selected to form the mounting rail 1. In the position of use, the central web portion 19 of the "I"-shaped member is in a vertical plane. Formed on the underside of the wall portion 20 of the upper flange of the "I"-shaped member, on both sides of the web portion 19 thereof, are two side plate portions 21. Bar portions 22 are formed at the lower ends of the two side plate portions 21, and project therefrom at right angles, more particularly in such a way that the free edges of the bar portions 22 are directed towards each other and define in the longitudinal direction of the mounting rail member, a corresponding gap in which the pins or rollers 23 of the chain 2 are disposed. The gap is of such a size that the rollers 23 of the chain 2 can be guided in a spaced condition therein without however undue transverse movement of the chain.

The side plates 24 of the chain 2 on both sides of the end surfaces of the rollers 23 project beyond the rollers in the direction of the roller diameters, so that they prevent the chain from

sliding out of the gap between the free edges of the bar portions 22, upwardly or downwardly. The side plates 24, which are on the top in the position of use, of the chain 2 which rotates in a horizontal plane in the position of use, lie on the side regions of the edges of the bar portions, so that the chain 2 is carried in such a way that it cannot sag, while the fact that the edges of the bar portions 22 engage the rollers prevents lateral movement and ensures that for example, a switch actuating member secured to the chain is moved into a correct actuating position within close limits in a three-dimensional mode.

It will be seen that when the above-described elongate "I"-shaped member or extrusion is used, the cost of forming the chain guide means is virtually negligible. By virtue of the chain being guided over virtually the entire length of the run of the chain, the chain is in a precise position everywhere, and accordingly there is no need for a special adjusting operation for chain guide means which have to be mounted individually. In principle, the mounting rail can be produced on its own, and a suitable shaped member for forming the chain run guide means can be mounted thereon, which guide member can then also comprise another material. The preferred form of one-piece manufacture in particular in the form of an extruded member can be manufactured conveniently and in accordance with the purposes involved herein, from an aluminium alloy.

CLAIMS

1. A draw chain drive arrangement, in particular a door drive arrangement, having a drive motor, a chain which is guided over a drive chain wheel driven by the motor, and a chain guide wheel, with a draw member fixed to the chain, and a mounting rail which holds the two chain wheels at a spacing and which is provided with a sliding guide means for the chain in the region of the runs of the chain between the chain wheels, characterised in that, for each run of the chain, the mounting rail has two bar portions which engage between the side plates of the chain towards the pins or rollers of the chain and which are arranged to extend in the longitudinal direction of the run of the chain at least approximately over the entire length thereof, in the region of the mounting rail.

2. A drive arrangement according to Claim 1, characterised in that the bar portions are portions of an elongate shaped member forming the mounting rail.

3. A drive arrangement according to Claim 2, characterised in that the bar portions are formed in such a way as to project perpendicularly on side plate portions which extend vertically in the position of use and which in turn are formed on a horizontally extending wall portion of the mounting rail.

4. A drive arrangement according to Claim 3, characterised in that the mounting rail is of an "I"-shaped configuration with a web portion which extends vertically in the position of use, and that the side plate portions are formed in such a

way as to project into the space between the flange portions of the "I"-shaped member.

5. A drive arrangement according to any one of the preceding claims, characterised in that the

5 mounting rail comprises an aluminium alloy.

6. A draw chain drive arrangement, substantially as hereinbefore described, with reference to the accompanying drawings.